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RADIATION ATTENUATION THROUGH WATER SPRAYS

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Content

- Introduction
- System layout and operating parameters
- Spray behaviour
- Thermal radiation
- Single and double water spray curtain
- Parametric analysis
- Operating parameters of the improved system
- Conclusions and recommendations



Introduction

Accident scenario of a fuel tank fire

- nearest building located 45 m from the fire
- thermal irradiation loading of 6 kW/m²





Figure 1: Layout of the fire source, water curtain and protected buildings

Introduction



Insufficient performance using a single water curtain array

An additional study is required to characterise and understand the main parameters of the radiation heat shielding system!



System layout and operating parameters



- The water distribution pipe and nozzles elevation of 5 m
- 21 water curtain nozzles directed upward (5 m spacing, 92 l/min per head, 160° spray cone angle)
- 26 flat fan heads directed downward
 (2.5 m spacing, 63 l/min per head, 120° spray cone angle)



Figure 3: Layout sprinklers

System layout and operating parameters

Two types of sprinklers used for the barrier:

- upper barrier: GW water curtain nozzle (www.gwsprinkler.com)
- lower barrier: PNR GX flat fan nozzle (www.pnr.eu/prodotti/gx)

Design and performance characteristics ambiguity of the utilised sprinklers



System layout and operating parameters

Additional data related to the system layout:

- Fuel tank diameter of 60 m
- Maximum irradiation heat flux of 20 kW/m²
- Distance of interest 75 m from the origin (i.e. 45 m from the tank)
- Water curtain location approx. 10 m from the buildings (i.e. 35 m from the tank)

External conditions:

- Temperature of 12°C
- 75% relative humidity
- Initial zero wind conditions



Spray behaviour

CFD simulations to determine the most likely size of droplets and their speed

- representative section of the water curtain
- water nozzles in the mid-elevations
- periodic boundary conditions



Figure 4: Simulation domain la rout (periodic section)

Spray behaviour

- 5° spray rotation to avoid droplet collision
- spray coverage with 100% overlap





Figure 5: Spray arrangement [1]



Spray behaviour

Steady-state CFD simulations with 10000 particle tracks

Comparison with observations

- droplet diameter of 800 μ m
- downward speed of 12 m/s
- upward speed of 20 m/s



Thermal radiation

Monte Carlo radiation model with 10 mil photon tracks

- Directional source equivalent to 20 kW/m² blackbody radiation at the tank wall
- Multigrey radiation model for the absorption and emission of the gaseous mixture
- Radiation absorptivity associated with spray droplets [2]:

 $\alpha_{\lambda} = 0.95[1 - exp(-0.875\xi_{\lambda}d)]$

where $\xi_{\lambda} = 4\pi k_{\lambda}/\lambda$ and d is the droplet diameter

Negligible effect of scattering



Thermal radiation

Attenuation of thermal radiation due to absorptivity (and emissivity) of

- liquid droplets produced by water sprays
- water vapour high initial relative humidity (75%) or droplet evaporation

Modelling tasks conducted in the cylindrical coordinate system



Thermal radiation

Radiation attenuation in absence of the spray curtain





Single and double water spray curtain

CFD simulations for a single and a double water spray curtain

- Particle tracks for the single water spray (Fig. 6)
- Particle tracks for a staggered arrangement of the double water spray curtain (Fig. 8)



Figure 8: Particle trajectories of the double water spray curtain



Single and double water spray curtain







Figure 9: Irradiation heat flux for the single water spray curtain

Figure 10: Irradiation heat flux for the double water spray curtain



Single and double water spray curtain

Averaged irradiated heat flux at the distance of 75 m from the origin:

- single water spray curtain: 4687 W/m²
- double water spray curtain: 3674 W/m²

Confidence in reducing the irradiation heat flux at the distance of 75 m below 3 kW/m²



Algebraic modelling of the radiation attenuation to determine the required droplet size

• Definition of the radiation intensity based on

$$\frac{dI}{dx} = -\{(1 - r_{drop})\kappa_{amb} + r_{drop}\kappa_{drop}\}I$$

where

 r_{drop} is volume fraction of droplets in the spray κ_{amb} and κ_{drop} are ambient and droplet absorption coefficients x is the radial distance from the source origin



- Calculation of the ambient absorption coefficient κ_{amb} using the Multigrey radiation model [11]
- Absorption coefficient of spray droplets [14]

$$\kappa_{drop} = \frac{1.5}{d} \alpha$$

where

lpha is the droplet radiation absorptivity d is the droplet diameter

Calculation of irradiation heat flux

$$q_{irrad} = \frac{I}{2\pi x}$$



Validation of the algebraic using the CFD result for a single spray curtain



Figure 11: Radial variation of the irradiation heat flux (algebraic model)



- Calculation of the radiation distribution (Fig. 11) with the algebraic model for droplet diameter d = 800 μm droplet volume fraction r_{drop} = 0.015 % spray width x_{spray} = 2.0 m
- Droplet input parameters obtained directly from CFD simulation results

Irradiation heat flux at the distance of 75 m from the origin:

- algebraic model: 4713 W/m²
- CFD simulation: 4687 W/m²



Parametric analysis using the developed algebraic model:

- droplet diameter reduced from 800 to 200 μm
- constant liquid water flow rate of 63.0 + 92.0 + 63.0 l/min over 5 m long section



Avoiding the model dependence on the input parameters from the CFD simulations:

• Calculation of droplet speed (*u*) from the force balance:

$$\frac{1}{2}C_D\rho u^2\left(\pi\frac{d^2}{4}\right) = \left(\rho_{liq} - \rho\right)g\left(\pi\frac{d^3}{6}\right)$$

where the droplet drag coefficient (C_D) is obtained via the Schiller-Neumann correlation

• Definition of the main attenuation parameter ($r_{drop} x_{spray}$) based on the mass conservation equation

$$\dot{m}_{liq} = \rho_{liq} u(r_{drop} x_{spray}) l_{sect}$$

where \dot{m}_{liq} is the mass flow rate and l_{sect} is the length of the section (i.e. 5 m)



Reduction in the droplet diameter (*d*) causes:

- absorption coefficient (κ_{drop}) increase
- droplet speed (*u*) decrease
- main attenuation parameter ($r_{drop} x_{spray}$) increase

The droplet terminal speed (*u*) may lead to an overestimate of the radiation attenuation!



d [µm]	u [m/s]	$r_{drop} x_{spray} [m]$	q _{irrad} [W/m ²]
800	3.04	2.39E-04	4085
700	2.69	2.70E-04	3653
600	2.33	3.12E-04	3075
500	1.95	3.74E-04	2304
400	1.55	4.70E-04	1333
300	1.12	6.47E-04	380
200	0.68	1.08E-03	6

 Table 1: Algebraic model – analysis results

- Desired level of irradiation for droplet diameter below 500 μm
- Increased number of smaller nozzles (with lower flow rate) per unit length



- CFD simulations for the droplet diameters of 500, 400 and 300 μm
- Doubling the number of nozzles & halving the water flow rate per nozzle

Heads directed upward:

- 1/2 · 5.0 m spacing
- nominal flow rate 1/2 · 92 l/min per head

Heads directed downward:

- 1/2 · 2.5 m spacing
- nominal flow rate 1/2 · 63 l/min per head



- Vertical elevation increase for the frame with nozzles from 5 to 6.5 m
- Unchanged initial droplet speed 12 m/s for the downward sprays, and 28 m/s for the upward sprays









Figure 13: Irradiation heat flux for droplet diameter of 300 μm





Figure 14: Vertical distribution of the main attenuation parameter $r_{drop} x_{spray}$

Figure 15: Vertical distribution of irradiation heat flux @ 75 m



Conclusions and recommendations

Average irradiation heat flux at 75 m from the origin

d [µm]	q _{irrad} [W/m ²]	
500	3577	
400	3020	
300	2184	

 Table 2: CFD results of the improved system

- Reduction of the droplet diameter to 400 μm for a single curtain system & raising the nozzles to 6.5 m
- Necessity of a double curtain system for larger droplet sizes



Thank you !



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